

DRIVE

## SERVOMATE ${ }^{\circledR}$ disc couplings

SERVOMATE ${ }^{\circledR}$ disc couplings have been specially designed for servomotor applications. The aluminium hubs and the compact design provide low mass moment of inertia resulting in a reliable and maintenance free coupling for high speeds. The double disk pack execution has been designed for applications with radial misalignment. Note: It is possible to have aligned keyways upon inquiry.


|  | $\begin{gathered} \mathrm{T}_{\mathrm{KN}} \\ {[\mathrm{Nm}]} \end{gathered}$ | $T_{K \text { max }}$ [ Nm ] | Torsional rigidity $\mathrm{C}_{\mathrm{T}}$ [ $\mathrm{Nm} / \mathrm{rad}$ ] |  | Max. speed [rpm] |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | GSM | GSMC |  |
| 0-6 | 20 | 40 | 12.000 | 6.000 | 16.000 |
| 0-6 | 30 | 60 | 30.000 | 15.000 | 12.000 |
| 0-6 | 60 | 120 | 60.000 | 30.000 | 10.000 |

*= with max bore.
**= prebored not in tolerance.

| Size | GSM misalignment |  |  | GSMC misalignment |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Radial <br> $[\mathrm{mm}]$ | Axial <br> $[\mathrm{mm}]$ | Angular <br> $\left[{ }^{\circ}\right]$ | Radial <br> $[\mathrm{mm}]$ | Axial <br> $[\mathrm{mm}]$ | Angular <br> $\left[{ }^{\circ}\right]$ |
| $\mathbf{1 5}$ | - | 0,5 | 1 | 0,16 | 1,0 | 2 |
| $\mathbf{2 0}$ | - | 0,6 | 1 | 0,25 | 1,2 | 2 |
| $\mathbf{2 5}$ | - | 0,8 | 1 | 0,30 | 1,6 | 2 |


| Size | Trasmissible torque [ Nm ] related to shaft diameter [ mm ] |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\varnothing 10$ | $\varnothing 11$ | $\varnothing 12$ | 014 | Ø15 | $\varnothing 16$ | 019 | $\varnothing 20$ | Ø22 | Ø24 | Ø25 | Ø28 | Ø30 | Ø32 | Ø35 |
| 15 | 20 | 22 | 24 | 28 | 30 | 32 | 38 | 40 | - | - | - | - | - | - | - |
| 20 | - | - | 24 | 28 | 30 | 32 | 38 | 40 | 44 | 48 | 50 | - | - | - | - |
| 25 | - | - | - | - | 55 | 59 | 70 | 73 | 81 | 88 | 92 | 103 | 110 | 117 | 128 |



| Coupling | GSM | 020 | MS | Screw tightening torque | Nm |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1 disc pack execution: GSM <br> 2 disc packs + spacer execution: GSMC |  |  | TKN | Coupling nominal torque | Nm |
|  |  |  | $\mathrm{T}_{\text {K max }}$ | Coupling maximum torque | Nm |
|  |  |  | $\mathrm{C}_{\mathrm{T}}$ | Torsional rigidity | Nm/rad |
| Size |  |  | J | Moment of inertia | $\mathrm{Kg} \cdot \mathrm{m}^{2}$ |
|  |  |  | W | Weight | kg |

## Selection in according to DIN 740.2

The coupling must be chosen so the applied working loads do not exceed the allowable values whatever the working conditions are.

## 1. Check the load with respect to the nominal torque

The nominal coupling torque must be greater than or equal to the nominal torque of the drive machine for all working temperatures.

$$
\mathrm{T}_{\mathrm{KN}} \geq \mathrm{T}_{\mathrm{N}} \cdot \mathrm{~S}_{\theta} \cdot \mathrm{S}_{\mathrm{D}}
$$

2. Check the load with respect to the torque peak values

The maximum coupling torque must be greater than or equal to the torque peaks that occur during operation for all working temperatures.

$$
\mathrm{T}_{\mathrm{KN}} \geq \mathrm{T}_{\mathrm{S}} \cdot \mathrm{~S}_{\theta} \cdot \mathrm{S}_{\mathrm{D}}+\mathrm{T}_{\mathrm{N}} \cdot \mathrm{~S}_{\theta}
$$

Motor-side peaks: $T_{S}=T_{A S} \cdot \frac{1}{m+1} \cdot S_{z}$
Driven-side peaks: $T_{S}=T_{L S} \cdot \frac{m}{m+1} \cdot S_{z}$
Or, in case of sporadic spikes: $\quad T_{K \text { max }} \geq T_{S} \cdot S_{\theta} \cdot S_{D}+T_{N} \cdot S_{\theta}$
If the peak does not cover the nominal $T_{N}$, ontribution, the $T_{N} S_{\theta}$ factors can be disregarded.

## Calculation coefficients

## $\mathbf{S}_{\theta}=$ Temperature factor

| $\mathrm{T}\left({ }^{\circ} \mathrm{C}\right)$ | $-30^{\circ} \mathrm{C} /+90^{\circ} \mathrm{C}$ |
| :---: | :---: |
| $\mathrm{S}_{\theta}$ | 1 |

## Starting frequency factor

| $\mathrm{S} / \mathrm{h}$ | $<20$ | $<60$ | $<120$ | $<180$ | $<240$ | $>240$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{~S}_{\mathrm{z}}$ | 1 | 1,2 | 1,4 | 1,6 | 1,8 | 2 |

$S_{D}=$ Torsional rigidity factor

| Tooling machines | Positioning <br> system | Speed and angular <br> acceleration indicator |
| :---: | :---: | :---: |
| 1,5 | 2 | $2,5 / 4$ |

Per macchine utensili - servomotori applicare 1,5-2.
$\mathbf{m}=$ Mass factor $=\frac{J_{A}}{J_{L}}$

